MATLAB-Simulink based Tool Development for Early Stage Design of Electric Powertrain during Conversion of Conventional Vehicle into Plug-In Hybrid Electric Vehicle

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ABSTRACT:

Electric Vehicles (EV) are becoming more popular due to an environment friendly approach. However, due to the dominance of fossil fuel-based conventional vehicle on road, emission reduction is a crucial task. In spite of continuous efforts to keep emission under control with alternative fuels, emissions are not under control. Conversion of Conventional Vehicle (CV) into Plug-In Hybrid Electric Vehicle (PHEV) is a promising solution to keep emissions under control of present vehicles running on the road as per norms. However, performance is limited by the size of an electric powertrain. Size of electric powertrain and its control strategy decided the fuel economy and emissions of vehicle i.e. the right size of the powertrain components is essential to fully exploit the benefits of the hybridization. The trend of electric powertrain design is from driving cycle. This will give better performance at that particular route only. Hence values are misleading for any other route and this design value cannot be generalized. The paper provides driving cycle independent generalized tool for the design of electric drivetrain during conversion of the conventional vehicle into the plug-in hybrid electric vehicle by fundamental force analogy method to get better performance of the vehicle. The MATLAB-Simulink tool is developed to size electric drivetrain parameters. The design parameters can be given as the input which will size the electric powertrain i.e. the size of motor and battery. In addition, the tool can be used to size battery of new PHEV. The obtained results have confirmed the effectiveness of proposed tool and compared to designed values of the vehicle. The size of electric powertrain can be utilized as early stage design during conversion of the conventional vehicle into the plug-in hybrid electric vehicle.

KEYWORDS: Electric Powertrain Plug-in Hybrid Electric Vehicle, Sizing, Software.

1. INTRODUCTION

PHEV is composed of an Internal Combustion Engine and Electric Machine (EM) which is usually powered by a battery through power electronics converter. Furthermore, the possibility to combine pure electric operation within no emission zones with the range capabilities of conventional vehicles makes converted PHEV the stepping stone towards the development of the electric mobility. Conversion of CV into PHEV is promising solution towards sustainable transportation for vehicles running on road, however there is less attention in terms of research on conversion of CV into PHEV as it is an ongoing challenge due to the limitation of space for fitting electric drivetrain, change in mechanical drivetrain components, size and weight constraint of electric drivetrain along with its energy management and control strategy. Few papers are published related to

conversion to HEV/PHEV mostly on conversion aspect and energy management strategy with stress on fuel economy [1-15]. Mostly electric powertrain is designed considering driving cycle only, which will give better performance only to considered driving cycle. There is no contribution in sizing of the electric drivetrain in a generalized way (sizing can be used for any driving route). As a matter of fact, PHEVs operation can be strongly affected by the selected component size. Therefore, once the mission of the vehicle is to the design of the electric powertrain with a fully electric mode for better performance of the converted plug-in hybrid electric vehicle (CPHEV) with a moderate range of the vehicle. Also, the design values should be driving cycle independent. The paper contributes towards MATLAB-Simulink based tool development of generalized electric powertrain design.

The paper is organized as below:

Design Parameter Input

Section 2 provides details of proposed MATLAB Simulink tool. Section 3 gives a methodology for modeling of power required to size electric powertrain with vehicle dynamic approach. In section 4 results are provided for sizing of electric powertrain during conversion of the conventional vehicle into the plug-in hybrid electric vehicle along with battery sizing for market available PHEV followed by conclusion in section 5.

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2. PROPOSED MATLAB-SIMULINK TOOL

The layout of the proposed tool can be seen in Fig. 1. It can be seen that design parameters are given as input at input side, and the size of motor and battery are outputs. The black box tool does the calculation required using fundamental vehicle dynamic approach by force analogy.

Sizing of Electric Powertrain



Fig. 1. MATLAB-Simulink Sizing Tool Layout.

The input parameters required are

- 1. Mass of vehicle in kg (including passenger and luggage).
- 2. The additional mass of the electric powertrain in kg.
- 3. The frontal area of the vehicle in m^2 .
- 4. The range of vehicle (electric) in km.
- 5. The speed of vehicle corresponding to base speed of the motor in km/hr.
- 6. Performance required (Acceleration details) e.g. 0 to 60 km/hr. in 10 secs. (Required speed in km/hr. and time in sec. to attain a required speed).

The output is the size of an electric drivetrain which is the size of the motor in kW and size of the battery in kWh.

3. METHODOLOGY

Electric powertrain is mostly sized from standard Driving Cycle (DC). Moreover, standard DC approach is misleading as it will give optimum performance for a specific route and the performance obtained cannot be generalized. Hence fundamental approach of the vehicle dynamic needs to be considered for the design of electric powertrain. Hence in this work for sizing tool development, the fundamental vehicle dynamic approach is used.

The minimum sizing of the electrical motor is done by comparing following powers modeled.

1. The power required for acceleration of the vehicle at the start is modeled by the power required to overcome inertia and rolling resistance. The power required to overcome rolling resistance at the time of starting is modeled using fundamental equations considering worst unpaved road condition. The inertia power is determined by modeling accelerating performance requirement of the vehicle.

2. The power required to overcome gradient is modeled by the power required to overcome rolling resistance and weight component for a low speed of the vehicle. The force required to overcome rolling resistance is modeled considering the normal value of the coefficient of rolling resistance and weight component is considering maximum grading

requirement condition.

3. The power needed at a base speed of the electric motor is modeled by considering force required to overcome rolling resistance and aerodynamic drag. The normal values of coefficient of rolling resistance and aerodynamic drag can be considered.

The motor sizing has done by comparing these three-power requirements and considering the efficiency of transmission between wheel and motor. The size of motor can be increased to improve regenerative capability with the limitation of space for fitting.

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The battery sizing solely depends on a range of vehicle per charge and the power corresponding to the speed of vehicle corresponding to the base speed of the motor.

When the default values are considered, design input requirements are;

 μ_{rmax} -coefficient of rolling resistance=0.05,

- ρ -air density in kg/m³ = 1.224,
- g- acceleration due to gravity in $m/s^2 = 9.81$.

The specification of two passenger car is considered for study as shown in Table 1 below.

| Vehicle | Hatchback [16] | SUV [17] |
|-------------------------|---|---|
| Gross Weight | 1400 kg | 1860 kg |
| Engine Type | 4 Cylinder Turbo Intercooled | 4 Cylinder |
| Displacement | 1405 cc | 2494 cc |
| Fuel Type | Diesel | Diesel |
| Maximum Power | 70 bhp @ 4500 RPM | 100.57bhp@3600rpm |
| Maximum Torque | 135 Nm @ 2500 RPM | 200Nm@1400-3400rpm |
| Mileage (ARAI) | 19.1 kmpl | 12.99 kmpl |
| Frontal area of vehicle | 2.12 m^2 | 2.632 m^2 |
| Battery | Li-ion | Li-ion |
| Motor | 3 Ph. Induction Motor | 3 Ph. Induction Motor |
| Base Speed | 60 km/hr. | 60 km/hr. |
| Range/charge | 60 km | 60 km |
| Acceleration | 1.667 m/s ² (0-60 km/hr. in 10 sec.) | 1.5873 m/s ² (0-100 km/hr. in 17.5 sec.) |

Table 1. Vehicle Specification of CPHEV.

4. RESULTS

The results of sizing of an electric powertrain for specifications of the vehicle are given in table 2.

Table 2. Early Stage Design Values.

| | - |
|-----------|---|
| Hatchback | Hatchback [16] |
| 23.66 kWh | 22.76 kWh |
| 16.56 kW | 15.78 kWh |
| SUV | |
| 30.26 kV | Wh |
| 21.26 kV | Wh |
| | Hatchback 23.66 kWh 16.56 kW SUV 30.26 kV 21.26 kV |

Note: Results are with consideration of the efficiency of battery-0.8, State of charge utilization of 0.7 and efficiency of transmission between motor and wheel as 0.8 as in [16].

In addition, the effect of variation in range and speed of vehicle corresponding to the base speed of motor on motor rating and battery capacity can be plotted.

The battery capacity in kWh for variation in range

per charge for a hatchback for the constant base speed of vehicle of 60 km/hr. is shown in Fig. 2.



Fig. 2. Battery Capacity in kWh vs. Range in km.

If the vehicle is running with rated power, then battery capacity increases with range as shown in Fig. 2.

The effect of variation in vehicle speed corresponding to the base speed of the motor on

minimum motor size and battery capacity is plotted, for a hatchback, as shown in Figs. 3 and 4.



Fig. 3. Motor Size in kW vs. Vehicle Speed in km/hr.

In addition, this software tool has tested for sizing of a battery of PHEV available in the market. The motor size is not possible as new PHEV is designed with smaller engine size with a higher value of motor rating. The parameters used for sizing of the battery are shown in Table 3.



Fig. 4. Battery Capacity vs. Vehicle Speed in km/hr.

It should be noted that the values given by company are under idle condition. Under reality, it is not possible to get the range for given value of the battery. In the actual case for the range given the value of battery capacity required is more.

| Table 3. Specifications of Market available PHEV. | | | | |
|---|----------------------------|----------------------------------|--------------------------------|--|
| Ford Fus | ion Energi-SE [18] 2017 | Honda Accord PHEV [20] 2017 | Mahindra e20 plus [22] 2017 | |
| Curb Weight | 1775 kg | 1723 kg | 937 kg | |
| Gross Weight | 1950 kg | 1900 kg | 1257 kg | |
| Frontal area of vehicle | 2.32 m^2 | 2.3 m ² | 2.12 m^2 | |
| Top Speed (projected) | 135 km/hr. | 100 km/hr. | 80 km/hr. | |
| Range/charge | 35 km | 21 km | 60 km | |
| Acceleration | 1.827 m/s^2 | 6.63 m/s ² | 1.97 m/s^2 | |
| (0-100 | km/hr. in 15.2 sec.) [19] | (0-100 km/hr. in 6.63 sec.) [21] | (0-60 km/hr. in 14.1 sec.) | |
| Battery capacity | 7.6 kwh | 6.7 kW | 13.44 kWh | |

| Table 3. Specifications o | Market available F | PHEV |
|---------------------------|--------------------|------|
|---------------------------|--------------------|------|

The parameters given above are used as input for a tool developed and the size of battery required are given in table 4.

| Tal | le 4. | Early | y Stage | Design | Va | lues |
|-----|-------|-------|---------|--------|----|------|
|-----|-------|-------|---------|--------|----|------|

| Battery Capacity | | |
|-------------------------|---------------|-----------|
| | Software tool | Actual |
| Ford Fusion Energi | 10.55 kWh | 7.6 kWh |
| Honda Accord PHEV | 9.971 kWh | 6.7 kWh |
| Mahindra e20 plus | 23.64 kWh | 13.44 kWh |

The default value of efficiency of the battery is considered as 80 %. However regenerative energy captured and SOC utilization range is not considered.

The value given by fundamental strategy is more as expected compared to given value. In addition, as per team BHP review, Mahindra Reva e20 plus tested extensively for two real-world driving patterns in typical Mumbai conditions [23]. The results are given below.

Table 5. Case 1/Day 1: Single Driver (with a lot of flyovers and mild climbs)

1 person - 80% of the time, 2 people - 20% of the time

Air-conditioning - OFF

50% of the driving was done at night with the headlights on

Entertainment screen was on for 30% of the time

Boost mode used for 25% of the driving

| oany Specified-140 km |
|-----------------------|
|) |

Table 6. Case 2/Day 2: Extremely rigorous (20% of the journey was spent going up and down steep hills)

3 people - 30% of the time, 2 people - 30% and 1 - 40%

Air-conditioner - ON for 100 % and 20 minutes of the A/C running when parked

Boost mode- used for 25% of the driving

Entertainment screen & music playing- ON for 100% of the time

Headlights- ON for 30% of the time

| | Range obtained: 60 kn | Company Specified-140 kn |
|--|-----------------------|--------------------------|
|--|-----------------------|--------------------------|

Hence the range of vehicle during the real-time driving cycle is only 87 km in case 1 and 60 km in case 2 compared to a specified range of 140 km on the website.

5. CONCLUSION

The early stage design tool using MATLAB-Simulink is demonstrated for sizing of battery and motor of an electric powertrain. In addition, plots of base speed and range variation on sizing can be studied by the proposed tool. The proposed tool has also used to size battery of new PHEV available in market. The obtained results show that the EV industry needs to be more mature regarding proving proper electric range for specified battery capacity as the values are misleading. The variation observed by real-time testing done by team BHP for Mahindra e20 plus is 62.14 % of specified range in case 1 and less than 50 % as low as 42.85 % of specified range in case 2. Hence driving cycle design of electric drivetrain trend should be shifted to a fundamental vehicle dynamic approach for the design of electric powertrain to get better performance of the vehicle under any driving condition.

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